(b) Orifice testing criteria. (1) Compressors in common use, as listed in the

following table, shall have orifice test criteria as follows:

Make	Compressor size	Single strokes per minute	Diameter of orifice (in inches)	Air pressure maintained (in pounds)
Westinghouse	9½	120	11/64	60
3	11	100	3/16	60
Westinghouse	150 CFM 81/2 CC	100	9/32	60
Westinghouse	120 CFM 8½	100	15/64	60
New York	2a	120	5/32	60
New York	6a	100	13/64	60
New York	5b	100	¹⁵ / ₆₄	60

Note: This table shall be used for altitudes to and including 1,000 feet. For altitudes over 1,000 feet the speed of compressor may be increased 5 single strokes per minute for each 1,000 feet increase in altitude.

(2) For compressors not listed in the table in paragraph (b)(1) of this section, the air pressure to be maintained shall be no less than 80 percent of the manufacturer's rated capacity for the compressor.

§ 230.72 Testing main reservoirs.

- (a) Hammer and hydrostatic testing. Except as described in paragraphs (b) through (d) of this section, every main reservoir, except those cast integrally with the frame, shall be hammer and hydrostatically tested during each annual inspection. The reservoir shall be hammer tested while empty and with no pressure applied. If no defective areas are detected, a hydrostatic test of MAWP shall be applied.
- (b) Drilling of main reservoirs. (1) Only welded main reservoir originally constructed to withstand at least five times the MAWP may be drilled over its entire surface with telltale holes that are \(^{3}\)\(^{1}\) of an inch in diameter. The holes shall be spaced not more than 12 inches apart, measured both longitudinally and circumferentially, and drilled from the outer surface to an extreme depth determined by the following formula:

$$D=(.6PR/(S-.6P))$$

Where:

- D = Extreme depth of telltale holes in inches but in no case less than one-sixteenth inch;
 P = certified working pressure in psi;
- $S = \frac{1}{6}$ of the minimum specified tensile strength of the material in psi; and
- R = inside radius of the reservoir in inches.
- (2) One row of holes shall be drilled lengthwise of the reservoir on a line intersecting the drain opening. When

main reservoirs are drilled as described in paragraph (b)(1) of this section, the hydrostatic and hammer tests described in paragraph (a) of this section are not required during the annual inspection. Whenever any telltale hole shall have penetrated the interior of any reservoir, the reservoir shall be permanently withdrawn from service.

(c) Welded main reservoirs without longitudinal lap seams. For welded main reservoirs that do not have longitudinal lap seams, an appropriate NDE method that can measure the wall thickness of the reservoir may be used instead of the hammer test and hydrostatic test required in paragraph (a) of this section. The spacing of the sampling points for wall thickness shall not be greater than 12 inches longitudinally and circumferentially. The reservoir shall permanently be withdrawn from service where the NDE testing reveals wall thickness less than the value determined by the following formula:

$$T=(PR/(S-.6P)$$

Where:

- t = Minimum value for wall thickness;
- P = Certified working pressure in psi;
- S = 1/5 of the minimum specified tensile strength of the material in psi, or 10,000 psi if the tensile strength is unknown; and
- R = Inside radius of the reservoir in inches.
- (d) Welded or riveted longitudinal lap seam main reservoirs. (1) For welded or riveted longitudinal lap seam main reservoirs, an appropriate NDE method that can measure wall thickness of the reservoir shall be used instead of, or in addition to, the hammer test and hydrostatic test. The spacing of the sampling points for wall thickness shall

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not be greater than 12 inches longitudinally and circumferentially. Particular care shall be taken to measure along the longitudinal seam on both plates at an interval of no more than 6 inches longitudinally. The reservoir shall be withdrawn permanently from service where NDE testing reveals wall thickness less than the value determined by the following formula:

T=(PR/(0.5S-0.6P))

Where:

- t = Minimum value for wall thickness;
- P = Certified working pressure in psi;
- S = 1/5 of the minimum specified tensile strength of the material in psi, or 10,000 psi if the tensile strength of steel is unknown; and
- R = Inside radius of the reservoir in inches.
- (2) Repairs of reservoirs with reduced wall thickness are prohibited.

§ 230.73 Air gauges.

- (a) Location. Air gauges shall be so located that they may be conveniently read by the engineer from his or her usual position in the cab. No air gauge may be more than 3 psi in error.
- (b) Frequency of testing. Air gauges shall be tested prior to reapplication following removal, as well as during the 92 service day inspection and whenever any irregularity is reported.
- (c) Method of testing. Air gauges shall be tested using an accurate test gauge or dead weight tester designed for this purpose.

§ 230.74 Time of cleaning.

All valves in the air brake system, including related dirt collectors and filters, shall be cleaned and tested in accordance with accepted brake equipment manufacturer's specifications, or as often as conditions require to maintain them in a safe and suitable condition for service, but not less frequently than after 368 service days or during the second annual inspection, whichever occurs first.

§ 230.75 Stenciling dates of tests and cleaning.

The date of testing and cleaning and the initials of the shop or station at which the work is done, shall legibly be stenciled in a conspicuous place on the tested parts or placed on a card displayed under a transparent cover in the cab of the steam locomotive.

§ 230.76 Piston travel.

- (a) Minimum piston travel. The minimum piston travel shall be sufficient to provide proper brake shoe clearance when the brakes are released.
- (b) Maximum piston travel. The maximum piston travel when steam locomotive is standing shall be as follows:

Type of wheel brake	Maximum piston travel (in inches)
Cam Type Driving Wheel Brake	3½ 6 8 9

§ 230.77 Foundation brake gear.

- (a) Maintenance. Foundation brake gear shall be maintained in a safe and suitable condition for service. Levers, rods, brake beams, hangers, and pins shall be of ample strength, and shall not be fouled in any way which will affect the proper operation of the brake. All pins shall be properly secured in place with cotter pine, split keys, or nuts. Brake shoes must be properly applied and kept approximately in line with the tread of the wheel.
- (b) Distance above the rails. No part of the foundation brake gear of the steam locomotive or tender shall be less than $2\frac{1}{2}$ inches above the rails.

§ 230.78 Leakage.

- (a) Main reservoirs and related piping. Leakage from main reservoir and related piping shall be tested at every 92 service day inspection and shall not exceed an average of 3 psi per minute in a test of 3 minutes duration that is made after the pressure has been reduced to 60 percent of the maximum operating pressure.
- (b) Brake cylinders. Leakage from brake cylinders shall be tested at every 92 service day inspection. With a full service application from maximum brake pipe pressure, and with communication to the brake cylinders closed, the brakes on the steam locomotive and tender must remain applied for a minimum of 5 minutes.
- (c) Brake pipes. Steam locomotive brake pipe leakage shall be tested at